

# HEX-AXIS HORIZONTAL MOVEMENT DYNAMIC SIMULATOR

## ~~BACKGROUND OF THE PRESENT INVENTION 1. Field of the Present Invention~~ This FIELD OF THE INVENTION

The invention relates to a hex-axis horizontal movement dynamic simulator, and more particularly to a 6-degree-s-of-freedom motion simulating equipment used in modular design.

## BACKGROUND ~~2. Description of Prior Art~~ Since the ~~OF THE RELATED ART~~

An early structure of thea 6-degree-s-of-freedom motion simulating platform in the early years was proposed by the Englishman Steward, this type of 6-degree of freedom motion simulating platform was and is customarily called the Stewart Platform. ——For a long time, there was no any significant breakthrough of improvement or invention in the mechanism design of the Stewart Platform, which was still confined in the traditional idea of employingemployed a hydraulic or pneumatic system to achieve the effect of changing the length of thean actuating rod by varying the stroke of a cylinder rod to enable a 6-degree-s-of-freedom spatial motion. Moreover, since the parts and components constructingcomprising the conventional Stewart Platform arewere not in modular in design, and the inherent drawback of oil and air leakage problemsoccasionally occurred

with the hydraulic and pneumatic system are always bother some that resulted in the inconvenience and disadvantage of replacing the parts and components in maintenance work. Therefore, the purposesystems, the Stewart Platform was inconvenient and required substantial maintenance.

#### SUMMARY OF THE INVENTION

A goal of the present invention is to provide a solution to the above-described problems of the conventional Stewart Platform as mentioned above by employing thea modular design as the core of the invention in steadinstead of the hydraulic or pneumatic system adoptedused by the conventional Stewart Platform.—

SUMMARY OF THE PRESENT INVENTION The major purpose Another goal of the present invention is to provide a hex-axis horizontal movement dynamic simulator whichthat can simulate the motion of 6 degrees of freedom without employing a hydraulic or pneumatic system.—The minor purpose

A further goal of the invention is to provide a hex-axis horizontal movement dynamic simulator having a modular structure whichthat comprises three modular movement control units of the same structure. The modular movement control units are located at the positions relative to each other forming three sides of an equilateral triangle and are pivoted to a load-carrying platform by means of thea universal-joint yoke mechanism corresponding to

each of the three movement control units to construct simulation platform for simulating the motion of 6 degrees of freedom. Another purpose.

A further goal of the invention is to provide a specific structure of the modular movement control unit whichthat can precisely control the movement of the load-carrying platform, and enableprovide a structure of motion simulation platform having 6 degrees of freedom.—

#### BRIEF DESCRIPTION OF THE DRAWING FIGURES DRAWINGS

FIG. 1 is the pictorial drawing showing the structure of the illustrates a first embodiment of the hex-axis horizontal movement dynamic simulator of the invention whichthat has three sets of movement control units of the same structure located separately at the positions forming three sides of an equilateral triangle.—

FIG. 2 is the schematic drawing of illustrates the hex-axis horizontal movement dynamic simulator shown in of FIG. 1 showing the variation of translation and angular motion of the load-carrying platform.—

FIG. 3 is a schematic drawing of the invention shown in FIG. 2 as viewed from otheranother direction.—

FIG. 4 is thea disassembly drawing showing the parts of the movement control unit shownillustrated in FIG. 1.— FIG. 5 is the pictorial drawing showing the structure of the second type of FIG. 5 illustrates a second embodiment of the hex-axis horizontal movement dynamic simulator of the invention having three sets of movement control unitss of the same structure located separately at the positions forming three sides of an equilateral triangle.—

FIG. 6 is thea disassembly drawing showing parts of the movement control unit shownillustrated in FIG. 5.—

FIG. 7 is the pictorial drawing showing the structure of theillustrates a third type of embodiment of the hex-axis horizontal movement dynamic simulator of the invention having three sets of movement control unitss of the same structure located separately at the positions forming three sides of an equilateral triangle.—

FIG. 8 is thea disassembly drawing showing the parts of the movement control unit shownillustrated in FIG. 7.—

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS INVENTION

Now referReferring now to FIG. 1 and FIG. 2, thea key point of the present invention is that no-any hydraulic or pneumatic system beingis used by thein a hex-axis motion simulator (10) is employed. One end of both10. For each universal-joint yoke

mechanism 27, one end of each of two fixed-length connecting rods {26} of fixed length are pivoted to a set of the universal-joint yoke mechanism {27}, and the other ends of the connecting rods {26} are separately connected to the transmission-joint yoke mechanism {25}. Further, the side each transmission-joint yoke mechanism {25} is pivoted to a set of sliding seat {24}, and the rectilinear translation motion and position of the each transmission-joint yoke mechanism {25} is controlled by controlling the rectilinear translation and position of the corresponding sliding seat {24} through which the connecting rod {26} is actuated so as to generate a motion of 6 degrees of freedom which in turn that controls the spatial motion and position of the load-carrying platform.—

The hex-axis horizontal movement dynamic simulator {10} as depicted in the invention comprises three movement control units {20} of the same structure which, that are fixed on the foundation {50} at the locations relative to each other forming three sides of an equilateral triangle. Each of the three movement control units {20} is pivoted to the load carrying platform {60} by means of a universal-joint yoke mechanism {27}. Thus the locations at which the three sets of universal-joint yoke mechanism {27} are pivoted to the load-carrying platform {60} form an equilateral triangle.—

Each movement control unit {20} is symmetrically structured to comprise a set of universal-joint yoke mechanism {27}, two connecting rods of fixed length {26}, two sets of transmission-joint yoke mechanism {s 25}, two sets of sliding seat {s 24}, two lead screws {23}, two sets of servo-driving mechanism {s 22}, and a set of rectilinear translation guide {21}. Since the universal-joint yoke mechanism {27} is pivotally connected to the load-carrying platform {60}, it can generate a motion of 1 degree of freedom relative to the load-carrying platform {60}; and s. Since one end of each of the two connecting rods {26} is pivotally connected to the same universal-joint yoke mechanism {27}, then the connecting rod can generate a motion of 2 degrees of freedom.—

Therefore, the end at which of the connecting rod is pivoted to the universal-joint yoke mechanism {27} has 3 degrees of freedoms for generating a spatial motion relative to the load-carrying platform.— Further, another the other ends of the two connecting rods are symmetrically pivoted to separate driving joint yoke mechanism {s 25} that can generate a spatial motion of 2 degrees of freedom also, s. Since the transmission-joint yoke mechanism {25} of each connecting rod {26} is pivoted to a set of sliding seat {24}, the transmission-joint yoke mechanism {25} has 1 degree of freedom for generating a spatial motion relative to the sliding seat {24}. —Therefore,

the end at which of the connecting rod {26} is pivoted to the transmission-joint yoke mechanism {25} has 3 degrees of freedom relative to the sliding seat {24} in making a for spatial motion also.—

Based on the above-mentioned arrangement, both ends of each connecting rod {26} of each movement control unit {20} all have 3 degrees of freedom for generating a spatial motion, and, because. Since all the connecting rods are fixed-length rigid body of fixed length bodies, therefore when the sliding seat {24} makes a rectilinear displacement is displaced rectilinearly, the transmission-joint yoke mechanism {25} on one end of the connecting rod {26} is restricted to do rectilinear translation together with the sliding seat {24} that, which enables the connecting rod {26} to generate a spatial displacement of 6 degrees of freedom, and t. Through the variation of spatial position of every connecting rod {26}, the universal-joint yoke mechanism {27} on anotherthe other end of the connecting rod {26} will generate a relative spatial-displacement following the rectilinear displacement of the transmission-joint yoke mechanism {25}.—

Therefore, when the sliding seat {24} makes a rectilinear translation to another place together with the transmission joint yoke mechanism {25} its to which it is pivoted to, the spatial position of the connecting rod {26} on the transmission-joint

yoke mechanism {25} associated with 6 degrees of freedom will vary. That i.e.s, the universal-joint yoke mechanism {27} on one end of each connecting rod {26} will change its spatial position relative to the transmission-joint yoke mechanism {25}, and actuate the load-carrying platform {60} to vary its spatial position.—Refer to

Each of FIGs. 1 through FIG. 3, corresponding 1-3 corresponds  
to the rectilinear displacement of different sliding seats {24} sliding, which slide on different movement control units {20} which that are respectively located at the positions forming three sides of an equilateral triangle, there will. The sliding seats 24 may have relative displacements spatially generated by for the load-carrying platform {60}. Therefore, through the synchronous and precise control of the rectilinear movement of each sliding seat {24} of each movement control unit {20}, such as the synchronous control of the rectilinear movement of each sliding seat {24} by may be provided by a computer system, the spatial movement of each set of universal-joint yoke mechanism {27} relative to each the others can be precisely controlled to achieve a precise control of the motion of load-carrying platform {60} and to generate linear and angular displacement spatially.—

In the present invention ~~t~~The mechanism which that enables each sliding seat {24} of each movement control unit {20} to generate a rectilinear motion comprises two lead screws {23}, two

sets of servo-driving mechanism {s\_22} including having servo-motor {s\_221}, a rectilinear translation guide {21} including having two guide seats {212} and two straight sliding rail {s\_211}. Each sliding seat {24} has a female screw thread which that engages with the lead screw {23}. The servo-motor {221} of the servo-driving mechanism {22} is employed to drive the lead screw {23} to rotate that, which enables the sliding seat {24} on one of the guide seats {212} of the rectilinear translation guide {21} to be guided by the straight sliding rail {211} and to generate a rectilinear displacement. Therefore, the rectilinear movement of each sliding seat {24} can be precisely controlled by the precise control of the rotating speed and angular displacement of the servo-motor {221} of each servo-driving mechanism {22}, through which a precise control of the variation of linear and angular displacement of the load-carrying platform {60} can be achieved.—

The first type of embodiment of the movement control unit {20} is shown in FIGs. 1 through FIG. 4, which 4 and comprises at least a base seat {40}, a set of universal joint yoke mechanism {27}, two connecting rods of fixed length {26}, two sets of transmission-joint yoke mechanism {s\_25}, two sets of sliding seat {s\_24}, two lead screws {23}, two sets of servo-driving mechanism {s\_22}, and a rectilinear translation guide {21}. —The base seat {40} is a longitudinal plate fastened to the foundation {50}

by bolt. The rectilinear translation guide {21} has two linear sliding rails {211} parallel to each other and two identical guide seats {212}. The two linear sliding rails {211} are installed on the surface of the base seat {40} along the longitudinal direction of the base seat and parallel to each other, and the bottom side of each guide seat {212} ~~are~~ has two parallel guide slots which ~~are prepared to~~ that match the shape and gauge of the two straight sliding rails {211}, t. Thus, each guide seat {212} can be installed on and match the two straight sliding rails {211}, and slide on the two straight sliding rails along the guiding direction.—

Each servo-driving mechanism {22} comprises a servo-motor {221} assembled with a driving pulley {222}, a driving belt {223}, a driven pulley {224} and a bearing plate {225} ~~which~~ that drives a lead screw {23}. The bearing plate {225} of each servo-driving mechanism {22} ~~are~~ is installed ~~in~~ at a position near both a ~~the~~ bracket for mounting the two lead screws {23} ~~by means of~~ with bearings ~~and~~ to have the two lead screws {23} parallel to the two straight sliding rails {211}. The driving pulley {222} is mounted on the driving shaft of the servo-motor {221}, and the driven pulley {224} is mounted on the lead screw {23}. The driving pulley {222} and the driven pulley {224} are connected by the driving belt {223}.—

The sliding seat {24} is in rectangular shaped and is fastened to the guide seat {212} of the rectilinear translation guide {21}. On the sliding seat {24} two penetrating holes {241}, {242} are prepared of which the h. Hole {241} has a female screw thread, and engages with the lead screw {23}, another. The other hole {242} is a passage for another lead screw {23} to pass through. Further, on the top surface of each sliding seat {24} is a mounting recess {243} for pivotally mounting the transmission joint yoke mechanism {25}.—

The transmission-joint yoke mechanism {25} comprises a U-shaped yoke {251} and a T-shaped pivot axis. The horizontal stub shaft formed on both sides of the T-shaped pivot axis are is pivoted to the two vertical portions of the U-shaped yoke {251} by means of a bearing and nut that enables the perpendicular stub shaft of the T-shaped pivot axis to have 1 degree of freedom of rotational motion relative to the U-shaped yoke {251}. Besides, On the bottom side of the U-shaped yoke {251} is a mounting shaft {253} which that is pivotally mounted to the mounting recess {243} by means of a bearing and nut that enables provide the transmission joint yoke mechanism {25} to have with 1 degree of freedom of rotatingrotational motion relative to the sliding seat {24} it's pivoted to. Or t The perpendicular stub shaft of the T-shaped pivot axis {252} of each transmission-joint yoke mechanism {25} has 2 degrees of rotatingrotational motion

relative to the sliding seat {24} it's to which it is mounted to.—

The universal-joint yoke mechanism {27} comprises an inversed U-shaped yoke {271}, a cardan shaft {272}, a neck-ring seat {274}, and a cover plate {275}. The left and right horizontal stub shafts formed on both sides of the cardan shaft {272} are pivoted to the two vertical portions of the inversed U-shaped yoke {271} by means of a bearing and nut etc. that enables the perpendicular stub shaft formed on the front and rear side of the cardan shaft {272} to have 1 degree of freedom of rotatingrotational motion relative to the inversed U-shaped yoke {271}, moreover, o. On the top side of the inversed U-shaped yoke {271} is a mounting shaft {273} which that is pivoted to the neck-ring seat {274} by means of a bearing, and a. A cover plate {275} is mounted on the upper side of the neck-ring seat {274}, through which the whole assembly of the universal-joint yoke mechanism {27} is mounted on the load-carrying platform {60}, t. Thus, the inversed U-shaped yoke {271} has 1 degree of freedom of rotatingrotational motion relative to the neck-ring seat {274} or the cover plate {275} it is pivoted to, or t. The perpendicular stub shaft on the front and rear side of the cardan shaft {272} of the inversed U-shaped yoke {271} has 2 degrees of freedom of rotatingrotational motion relative to the neck-ring seat {274} or cover-plate {275}.—

Every connecting rod {26} has a fixed length. On both ends of the connecting rod {26} are pivoting holes through which the front end of the connecting rod is pivotally connected to the front perpendicular stub shaft or rear perpendicular stub shaft of the cardan shaft {272} of the universal-joint {27} that enables. Thus, the pivoting hole on the front end of the connecting rod {26} has 1 degree of freedom of rotatingrotational motion relative to the perpendicular stub shaft of the cardan shaft {272} it's pivoted to, or, t. The pivoting hole on the front end of every connecting rod {26} has 3 degrees of freedom of rotatingrotational motion relative to the neck-ring seat {274} or cover plate {275}; Besides, t. The pivoting hole on the rear end of every connecting rod {26} is pivotally connected to the perpendicular stub shaft of the T-shaped pivot axis {252} by means of a bearing and nut that enablesprovide the pivoting hole on the rear end of every connecting rod {26} to have with 1 degree of freedom of rotatingrotational motion relative to the perpendicular stub shaft of the T-shaped pivot axis {252} it's pivoted to, or, t. The pivoting hole on the rear end of every connecting rod {26} has 3 degrees of freedom of rotatingrotational motion relative to the sliding seat {24} it belongs to.—

Since each end of the connecting rod {26} has 3 degrees of freedom of rotatingrotational motion, the whole connecting rod {26} has 6 degrees of freedom for generating a spatial motion. =

The above-mentioned mechanism, as verified by the equation of mobility in Spatial Mechanism, is proved to be above to generates a spatial motion of 6 degrees of freedom i.e. according to Gruebler's formula for a spatial mechanism:

$$F = 6(L - j - 1) + \sum_{i=1}^j f_i$$

$$L = 32, j = 36, \sum_{i=1}^j f_i = 36; F = 6$$

Where

- F: Number of degrees of freedom of the whole mechanism—
- L: Total number of members in the mechanism—
- J: Total number of joints in the mechanism—
- $f_i$ : The number of degree of freedom of the  $i^{th}$  joint.—

Therefore, the relative rotating angle and rotating speed of the servo-motor {221} of the servo-driving mechanism {22} of each movement control unit {20}, based on the required data or condition of the relative motion of the load-carrying platform {60} in space and by applying the precise calculation and control of the computer system (not shown in drawings), can be synchronously controlled, and the sliding seat {24} and transmission-joint yoke mechanism {25} on each of the three movement control units can synchronously generate different

rectilinear movements to drive the connecting rod-(s 26) to generate relative spatial-displacement tos and control the relative spatial-movement of each universal-joint yoke mechanism {27}, thus enableenabling the load-carrying platform {60} to generate different movement to attain the purpose of varying vary its of posture and angular position that canso as to simulate the state of a carrier (such as vehicle, ship, airplane and roller coaster etc.) in-making a spatial motion of 6 degrees of freedom.—

In the following, is another embodiment of the movement control unit {20} which that has the same mechanical structure ofand the same effect as that of the first type embodiment of the movement control unit {20} as depicted in the above, and. This second embodiment applies the same technical means and actuating principle to enable the load-carrying platform {60} to simulate a spatial motion of 6 degrees of freedom. However, t The construction members and the inter-actuating relationship can be obtained by reference to the detailed description of the first type embodiment mentioned above, which shall not be repeated here, t. The following description is only fordescribes the structure of other type ofsecond embodiment of the movement control unit {20}.—

The second type of embodiment of the movement control unit {20} is shown in FIGS. 5 and FIG. 6 whichand comprises at least a

machine bed {41}, one set of universal-joint yoke mechanism {27}, two fixed-length connecting rods {26} of the fixed length, two sets of transmission-joint yoke mechanism {s 25}, two sets of sliding seat {s 24}, two lead screws {23}, two servo-driving mechanism {s 22}, and a rectilinear translation guide {21} in which t. The components of the universal-joint yoke mechanism {27}, the connecting rod {26}, the transmission-joint yoke mechanism {25}, the lead screw {23}, the servo-driving mechanism {22}, and the rectilinear translation guide {21} are all the same as those in the aforesaid first type of embodiment of the present invention.—

But, the machine bed {41} of the second type of embodiment of the invention is a rectangular stand made of a metal plate having an inversed U-shaped cross-section which that is fastened on the foundation {50}, the. A cover plate {411} is mounted on both of the left and right ends of the machine bed {41} having with holes and an opening prepared on at appropriate position, ts. The servo-motor {221} of the servo-driving mechanism {22} is installed inside the machine bed {41}. The driving shaft of the servo-motor {221} extends to the outside of the machine bed {41}. Through the opening on the cover plate {411} of the machine bed {41}, a driving pulley {222} is mounted and fastened on the driving shaft of the driving-servo motor {221}. Two support plates {227} of the servo-driving mechanism {22} are installed at the places closed to

both ends of the machine bed {41} to form the support for pivotally mounting the two lead screws {23} by means of bearings in a position parallel to the two straight sliding rails {211} of the rectilinear translation guide {21}. = The driven pulley {224} is mounted and fastened on the lead screw {23} with a transmission belt installed on and passing through the driving pulley {222} and driven pulley {224}. = Therefore, the driving power of the servo-motor {221} is transmitted to the lead screw {23} through the driving pulley {222}, the transmission belt {223}, and the driven pulley {224}. —

The sliding seat {24} employed in the second type of embodiment of the invention comprises at least a sliding block {244} and a neck ring seat {246}. = The sliding block {244} is fastened on the guide seat {212} of the rectilinear translation guide mechanism {21}. = On the sliding block {244}, two holes are provided, one of which has a female screw thread, and engages with a lead screw {23}, another. The other hole serves as the passage for another lead screw to pass through; t. = The neck-ring seat {246} is fastened on the top side of the sliding block {244}, or, a fastening plate {245} shall be is installed on the top side of the sliding block {244} at, first, and then fasten the neck-ring {246} is fastened on the fastening plate {245}. = The mounting shaft {253} of the U-shaped yoke {251} of the transmission-joint yoke mechanism {25} is pivoted to the circular access on the top

side of the neck-ring seat {246} by means of a bearing and related parts.—

What shall be mentioned here is that ~~t~~The sliding seat {24} employed in the second type of embodiment and the same in the first embodiment can be exchanged and used in either of the two types of embodiment, or in other embodiments of the invention.—

The third type of embodiment of the movement control unit {20} is shown in FIGS. 7 and FIG. 8 which compresses at least and comprises a base seat {40}, a set of universal-joint yoke mechanism {29}, two fixed-length connecting rods {26} of fixed length, two sets of sliding yoke mechanism {s 28}, two leading screws {23}, two sets of servo-driving mechanism {s 22}, and a rectilinear translation guide {21} wherein the said. The connecting rod {26}, lead screw {23}, servo-driving mechanism {22}, and rectilinear translation guide {21} are the same as those employed in the first type of embodiment, and t. The structure of the universal-joint yoke mechanism {29} is similar to the sliding yoke mechanism {28}.—

The universal-joint yoke mechanism {29} of the aforesaid third embodiment comprises an inverted U-shaped yoke assembly {291}, a pivoting plate {293}, a pivoting shaft {295}, two fixing blocks {296}, an L-shaped yoke plate {297}, a fastening yoke plate {298}, and two cover plates {299}. The L-shaped yoke plate is formed by a horizontal portion and a vertical portion. The

horizontal portion is fastened on the load-carrying platform {60}, and on the. A vertical portion hole is provided for mounting a shaft. The fastening yoke plate {298} is a plate-shaped member with appropriate thickness having an appearance symmetric to that of the vertical portion of the L-shaped yoke plate {297}. A shaft mounting hole is also provided on the fastening yoke plate {298} which is to be assembled with the L-shaped yoke plate {297} to form a yoke assembly. The pivoting plate {293} is in rectangular shape with a pivoting access in its center position and horizontal stub shafts {294} extended symmetrically from both sides opposite to each other which arethat pivotally mounted in the hole on the L-shaped yoke plate {297} and the fastening yoke plate {298} by means of bearings and related parts, t. The two cover plates are fastened on one side of the vertical portion of the L-shaped yoke plate {297} and the fastening yoke plate {298} to fix the whole assembly that enablesand provide the pivoting plate {293} to have with 1 degree of freedom of rotatingrotational motion relative to the L-shaped yoke plate {297} and the fastening yoke plate {298} its fastened to. The yoke assembly {291} has a mounting shaft {292} extended upwardly from its top side and is mounted in the pivoting access in the center position of the pivoting plate {293} by means ofa bearing, and a cover is fastened on the mounting surface of the pivoting plate to fix the assembly. Therefore, the yoke assembly has 1 degree of freedom of

rotatingrotational motion relative to the pivoting plate 293 its pivoted to, orand has 2 degrees of freedom of motion relative to the L-shaped yoke plate {297} and the fastening yoke plate {298};The bottom side of the two flanks of the yoke assembly {291} hasa semicircular recess, and the fixing block {296} also hasa corresponding semicircular recess on the top side. A shaft {295} is pivotally installed by fixing the two fixing blocks on the bottom side of the two flanks of the yoke assembly {291}, and both ends of the pivoting shaft {295} can be pivotally connected to the connecting rod {26} that enables so as to provide the pivot hole on the front end of each connecting rod haswith 1 degree of freedom of rotatingrotational motion relative to the yoke assembly {291} it's pivoted to, or, hasand 3 degrees of freedom of rotatingrotational motion relative to the L-shaped yoke plate {297} and the fastening yoke plate {298}.—

The sliding yoke mechanism {28} employed in the third embodiment comprises a U-shaped yoke assembly {281}, a pivoting plate {283}, a shaft {285}, two fixing blocks {286}, an L-shaped sliding yoke plate {287}, a sliding fastening plate {288}, and two cover plates {289}.The saidL-shaped sliding yoke plate {287} has a horizontal portion and a vertical portion, and is fastened on the guide seat {212} of the rectilinear translation guide {21} through its horizontal portion. Besides, tTheL-shaped sliding yoke plate {287} has two penetrating holes, one of which hasa

female screw thread, and engages with the lead screw {23}, another. The other hole serves as a passage for another lead screw {23} to pass through. In addition, the vertical portion of the L-shaped sliding yoke plate {287} has a pivoting hole. The sliding fastening plate {288} is a plate-shaped member with of appropriate thickness having and has an appearance symmetric to that of the vertical portion of the L-shaped sliding yoke plate. Two penetrating holes and a pivoting hole are provided on the sliding fastening plate {288} of which t. The two penetrating holes are for the two lead screws {23} to pass through. A yoke assembly is formed by assembling the sliding fastening plate {288} and the L-shaped sliding yoke plate {287}. The pivoting plate {283} is in rectangular shaped with a pivoting access in a center position and horizontal stub shafts {284} extended symmetrically from both sides opposite to each other which are that pivotally mounted in the hole on the L-shaped sliding yoke plate {287} and the sliding fastening plate {288} by means of a bearing and related parts, and then t. Two cover plates are fastened on one side of the vertical portion of the L-shaped sliding yoke plate {287} and the sliding fastening plate {288} to fix the whole assembly so that enables the pivoting plate {283} has 1 degree of freedom of rotatingrotational motion relative to the L-shaped sliding yoke plate {287} and the sliding fastening plate {288} it's pivoted to. The yoke assembly {281} has a mounting shaft

{282} extended downwardly from its bottom side which that is pivotally mounted in the pivoting access in the center position of the pivoting plate {283} by means of a bearing and the related parts, and a. A cover is fastened on the pivoting plate {283} to fix the assembly, therefore. Thus, the U-shaped yoke assembly {281} has 1 degree of freedom of rotatingrotational motion relative to the pivoting plate {283}, or and has 2 degrees of freedom of rotatingrotational motion relative to the L-shaped sliding yoke plate {287} and the sliding fastening plate {288}. The top side of the two vertical portions of the U-shaped yoke assembly has a semicircular recess, and the a corresponding semicircular recess is also provided on the fixing block {286} on the bottom side. A shaft {285} is pivotally installed by fixing the two fixing blocks {286} on the top side of the U-shaped yoke assembly, and both ends of the pivoting shaft {285} can be pivotally connected to the connecting rod {26} that enables so as to provide the pivot hole on the rear end of the connecting rod {26} has with 1 degree of freedom of rotatingrotational motion relative to the U-shaped yoke assembly {281}, or has and 3 degrees of freedom of rotatingrotational motion relative to the L-shaped sliding yoke plate {287} and the sliding fastening plate {288}. Since either of the two both ends of the connecting rod {26} has ve 3 degrees of freedom of rotatingrotational motion, each

connecting rod +26) shall haves 6 degrees of freedom of  
rotating rotational motion in space.